

**Design Conditions:**

Code: **ASME B31.3**  
Year: **2018**  
Addenda: -  
MAWP Internal: **2337** psi  
MAWP External: **0** psi  
Max. Temperature: **374** °F  
MDMT: **32** °F  
MDMT Pressure: **2337** psi  
Corrosion Allowance: **0** in  
Hydrostatic Test: **N/A** psi  
Impact Testing: **None**  
Impact Exemption: **323.2.2**  
Radiography: **None**

**Conclusion:**

The Piping w/Spring Hanger Analysis Sample Report has been analyzed to ASME B31.3, 2018 edition rules and is found acceptable.

**ASME B31.3 Piping Stress Analysis**

Cust: **PVEng**  
File: **16579psa-1 R0**  
Desc: **Piping w/Spring Hanger Analysis Sample Report**  
Dwg: **General Arrangement Drawings**

Date: **January 28, 2021**



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Reviewer: **Martin Hopgood, P.Eng.**



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**Scope of work:**

This document presents the input data and stress analysis results for piping installed in Canada. The piping system installed and analysed in this report is comprised of the following components (see Figure 1):

- NPS 4" SCH 80 Inlet (Interface point: N4)
- NPS 4" SCH 80 Outlet (Interface point: N5)

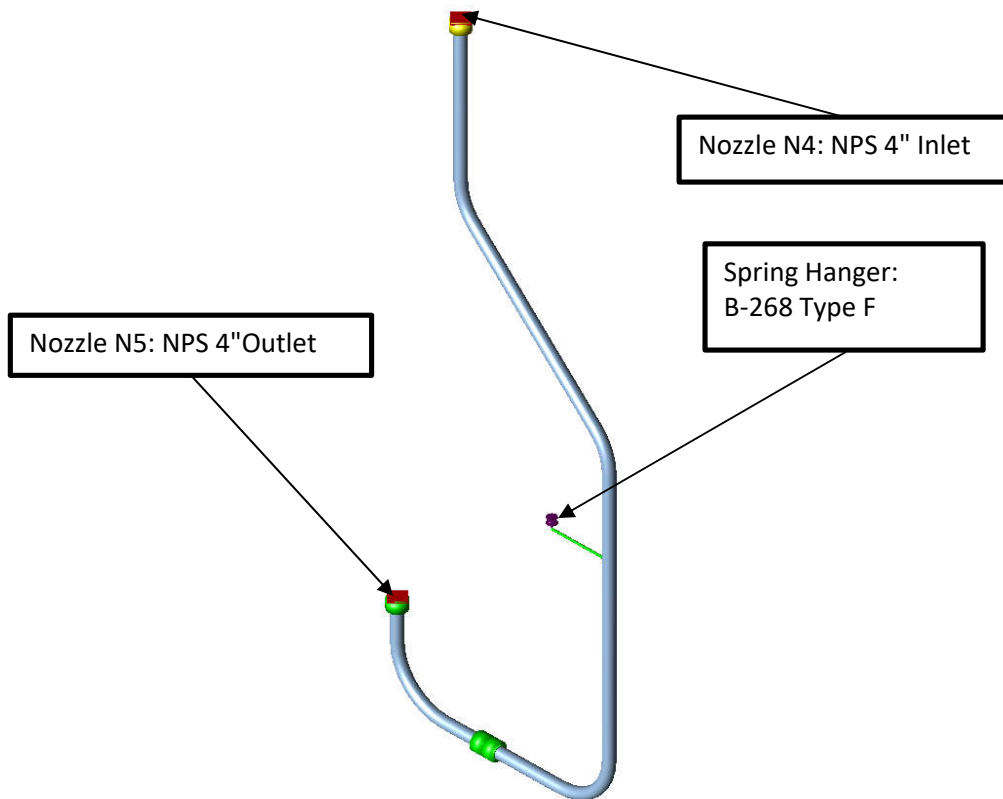


Fig-1: NPS 4" Connecting Inlet and Outlet Piping

The following sections present the analysis methodology, considerations and results. Applicable references are presented in Section "References".

**Goal:**

In accordance with the rules of ASME B31.3, Chapter II, Part 5, a piping flexibility (stress) analysis (PSA) is conducted to validate the design. The PSA is performed in accordance with ASME B31.3 rules and evaluated at the code allowable stresses for each load case.

**Summary Conclusions:****Analysis Software**

CAESAR II v.12 Build 20040

**Analysis Type**

A static linear elastic study is performed using small displacement theory.

**Materials**

Material properties used in this report are obtained from ASME B31.3, 2018 Edition. Stress limits are set in accordance with the B31.3 code rules.

**Model Information, Restraints & Loads**

A model of the piping system was created in Caesar II using the client provided drawings and design data. Conservatively it was assumed that the Nozzles N4 and N5 are fully restrained. The loads setup values for the existing Spring Hanger were not provided, the client requested PVEng to make recommendations on Spring Hanger operating load value, size and type. The Spring Hanger location was determined by client and it will be located at 25.59 in (650 mm) in -X direction supported by trunnion. Trunnion will be designed by others and is not part of this analysis report. The internal design pressure of 2,337 psi @ 374 °F was applied to all elements. The fluid media is urea solution.

**Results**

In summary, all the analysis results and observed stresses satisfied the evaluation criteria and are below their respective allowable limits. With the analysis methodology, considerations and results presented in this document, the piping system is deemed structurally adequate for the specified design purposes.

**Recommendations**

Recommended Spring Hanger to be used in this application based on CAESAR II Output vales is:  
Anvil B-268, Type F, Size 8

**Analysis Conclusion:**

The Piping w/Spring Hanger Analysis Sample Report has been analyzed to ASME B31.3, 2018 edition rules and is found acceptable.



## Analysis Type

A piping flexibility analysis was performed according to ASME B31.3, 2018 Edition rules using CAESAR II v.12 Build 200403 software. All documents were received electronically. Communication with and documentation supplied by the client provided the basis for the data used in the analysis. The data is summarized as follows:

## Analysis Data

Design Conditions:

Design Pressure: 2,337 psi

Design Temperature: 374 °F

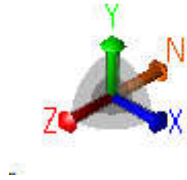
Material and Restraint types along with their general locations were indicated on the piping layout general arrangement drawings.

Seismic load was applied as uniform seismic horizontal acceleration in accordance with NBC-2015

Piping system is located indoor, Wind and Snow load is not applicable.

The piping stress model was built to analyze the system and meet all the necessary requirements.

The coordinate system in CAESAR II is shown below. The N - North direction is arbitrary as no plant N direction was provided.



### Elements and Nodes

Beam elements were used based on the differential length between nodes. Each element has two nodes represented as FROM and TO.

Node numbers are referred as N- # where required.  
Element length units are inches.

### Load Nomenclature

W = Dead Weight  
T1 = Design Temperature  
T2 = MDMT Temperature  
P1 = Design Pressure  
H = Spring Hanger Loads  
U1 = Seismic Load " X" direction  
U3 = Seismic Load " Z" direction

### Load Case Nomenclature

EXP = Expansion (Thermal Only)  
SUS = Sustained (Pressure and Weight)  
OPE = Operating (SUS + Thermal Load + Spring Hanger Load)  
OCC = Occasional (Seismic Only; SUS + Seismic)

### Units

F = Force (Units =lb)  
M = Moment (Units = lb-ft)  
D = Displacement (Units = in)





**Design Code :** ASME B31.3, 2018 Edition  
**Design Temperature :** 374 °F  
**Design Pressure :** 2,337 psi  
**MDMT :** 32 °F  
**Fluid Service :** Urea Solution No Lethal

**Material and Specification Data**

Material was determined from the drawings with allowable stresses taken from ASME B31.3 Appendix A. Tolerances and physical properties are taken from ASME BPVC Section VIII-1 Part II. The following table summarizes the information in the model.

<b>Component</b>	<b>Material</b>	<b>Size (NPS)</b>	<b>Thickness / Rating</b>	<b>Corrosion Allowance</b>
Pipe	A-790 Pipe S32906	4	Sch. 80	None
Flanges	A105	4		None

Figure 2 shows the ISO piping model.



Fig-2. Piping Model

Figure 3 shows all the nodes ID used in the model.

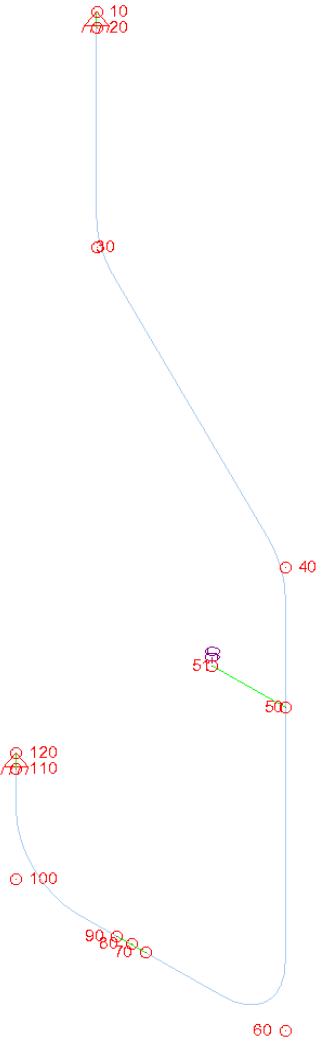


Fig-3. Model Nodes Labels

Figure 4 shows the material assignment.

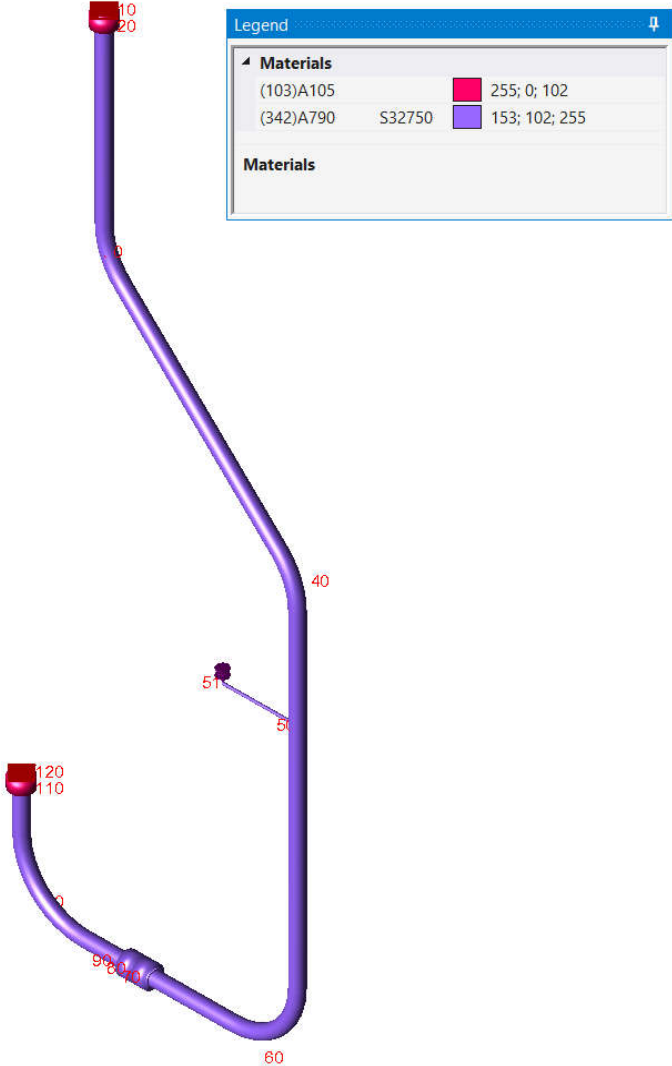


Fig-4. Model Material Assignment

Figure 5 show insulation assignment.

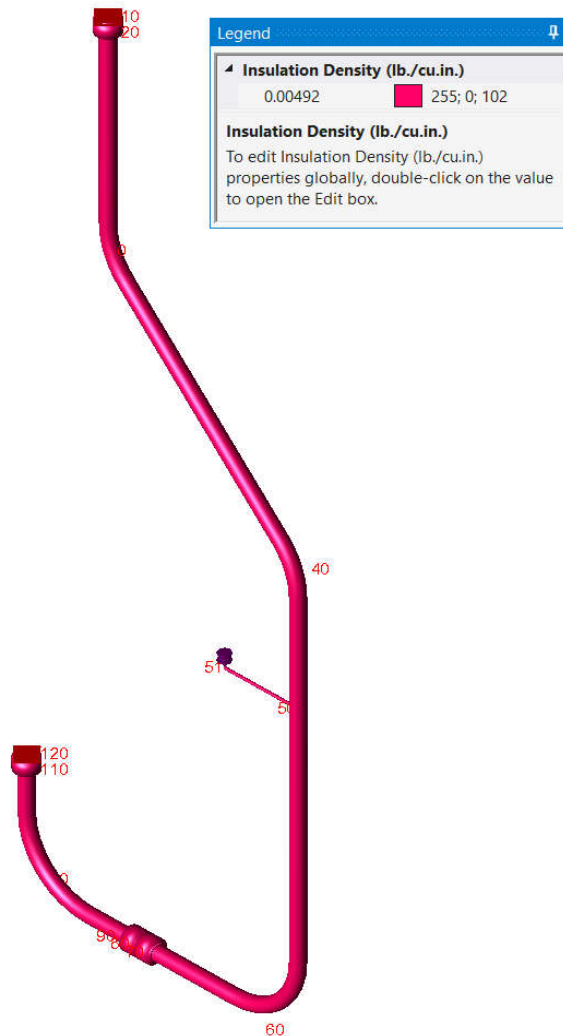


Fig-5. Model Insulation Assignment

Self weight of pipes has been modeled by assigning material and insulation density to each component as applicable. All flanges are modeled as rigid element with appropriate weight assigned. Client provided Insulation design data were thickness of 3" Mineral Wool with density of 0.00492 lb/in<sup>3</sup> is used in the model.

Assignment of Design Temperature T1 is to simulate the behavior of the pipeline at fluid design temperature and is shown on Figure 6.

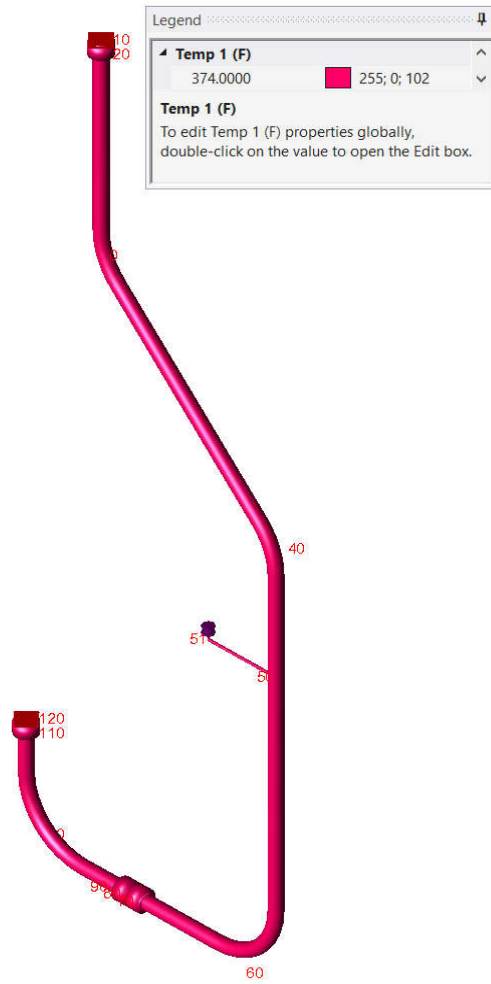


Fig-6. Model Design Temperature T1 Distribution.



Assignment of corresponding Design Pressure P1 for Design Temperature T1 and is shown on Figure 7.

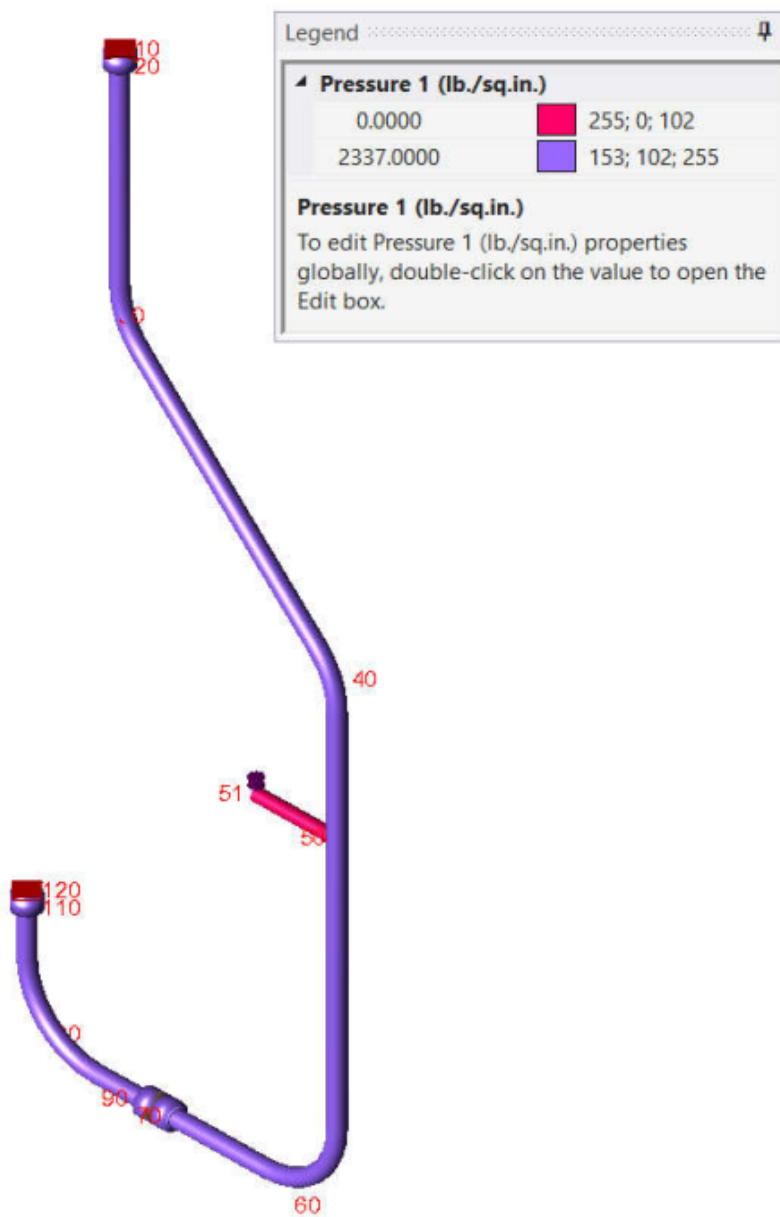


Fig-7. Model Design Pressure P1 Distribution.

Seismic Load per NBC-2015 was used in the model.

Seismic Horizontal Acceleration for non-structural components and equipment ( NBC-2015; 4.1.8.18)

Importance Factor:  $I_e = 1.3$  Provided by client - NBC-2015; Table 4.1.8.5

Component Factor:  $C_p = 1.5$  NBC-2015; Table 4.1.8.18; Category 16

Component Response Factor:  $R_p = 1.25$  Provided by client

Component Force Amplification Factor:  $A_r = 1.0$  NBC-2015; Table 4.1.8.18; Category 16

Spectral acceleration:  $S_a(0.2) = 0.086$

Site Class: D

Seismic Horizontal Acceleration for non-structural components and equipment ( NBC-2015; 4.1.8.18):  $a_h = 0.106 \text{ (m/s}^2\text{)}$

Loads combination defined and analysed in Caesar II model are shown on Figure 8 below.

	Exclude ▾	Definition ▾	Name ▾	Stress Type ▾
L1	<input type="checkbox"/>	W	WEIGHT FOR HANGER LOAD	HGR
L2	<input type="checkbox"/>	W+T1+P1	OPERATING FOR HANGER LOAD CASE CONDITION	HGR
L3	<input type="checkbox"/>	W+T1+P1+H	OPERATING CASE CONDITION	OPE
L4	<input type="checkbox"/>	W+P1+H	ALTERNATE SUSTAINED STRESS BASED ON OPERATING	SUS
L5	<input type="checkbox"/>	W+P1+H	SUSTAINED CASE CONDITION	SUS
L6	<input type="checkbox"/>	W+T2+P1+H	MDMT OPERATING CASE CONDITION	OPE
L7	<input type="checkbox"/>	W+T1+P1+H+U1	OPP+EX	OPE
L8	<input type="checkbox"/>	W+T1+P1+H-U1	OPP-EX	OPE
L9	<input type="checkbox"/>	W+T1+P1+H+U3	OPP+EZ	OPE
L10	<input type="checkbox"/>	W+T1+P1+H-U3	OPP-EZ	OPE
L11	<input type="checkbox"/>	L7-L3	+EX ONLY	OCC
L12	<input type="checkbox"/>	L8-L3	-EX ONLY	OCC
L13	<input type="checkbox"/>	L9-L3	+EZ ONLY	OCC
L14	<input type="checkbox"/>	L10-L3	-EZ ONLY	OCC
L15	<input type="checkbox"/>	L11+L5	EX+SUS	OCC
L16	<input type="checkbox"/>	L12+L5	-EX+SUS	OCC
L17	<input type="checkbox"/>	L13+L5	EZ+SUS	OCC
L18	<input type="checkbox"/>	L14+L5	-EZ+SUS	OCC
L19	<input type="checkbox"/>	L7,L8,L9,L10	OPP + E (MAX)	OPE
L20	<input type="checkbox"/>	L15,L16,L17,L18	E+SUS (MAX)	OCC
L21	<input type="checkbox"/>	L3-L5	EXPANSION CASE CONDITION	EXP
L22	<input type="checkbox"/>	L3-L6	FULL EXPANSION CASE CONDITION	EXP

Fig-8. Load Combinations used in the analysis.



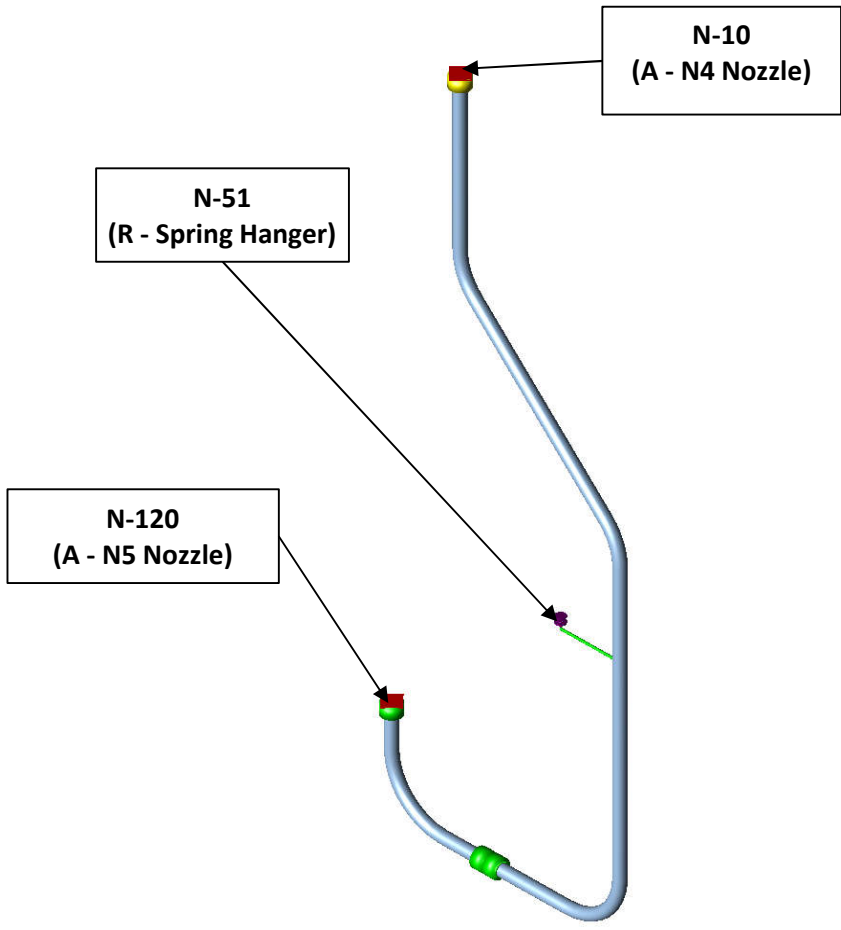


Fig-9. System Analysis Model

# Maximum Code Stresses

The maximum code stress for SUS; SUS+OCC and EXP cases were evaluated and are below the code allowable stress and less than 30% of allowable code stresses. The maximum code stresses are shown in Figure 10.

- SUS Maximum : N-50 @ 8186.3 (22.8%) (L5)
- SUS + OCC Maximum : N-50 @ 8395.9(17.6%) (L20)
- EXP Maximum : N-100 @ 5932.6 (13.6%) (L22)

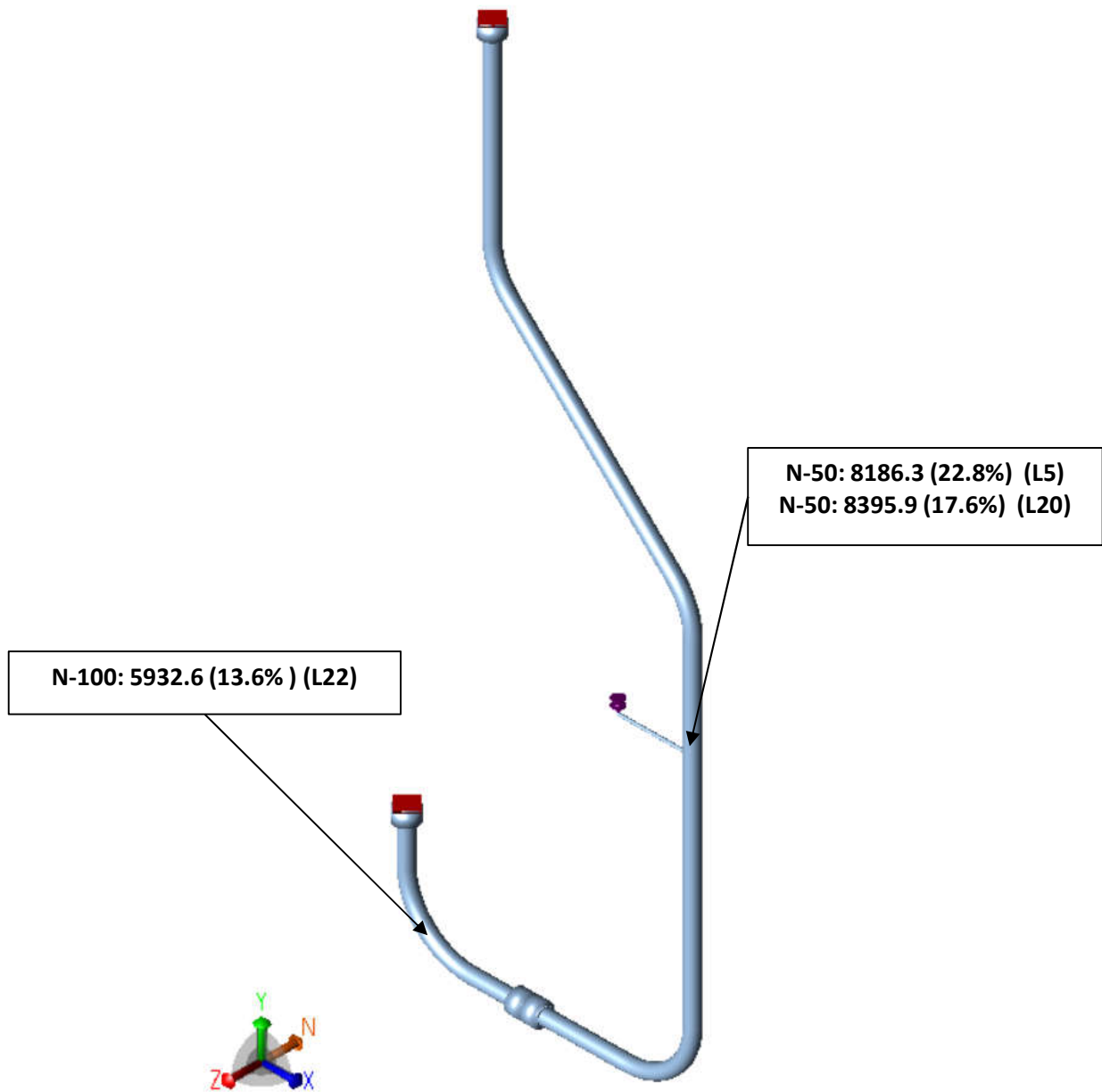


Fig-10. Maximum code stresses

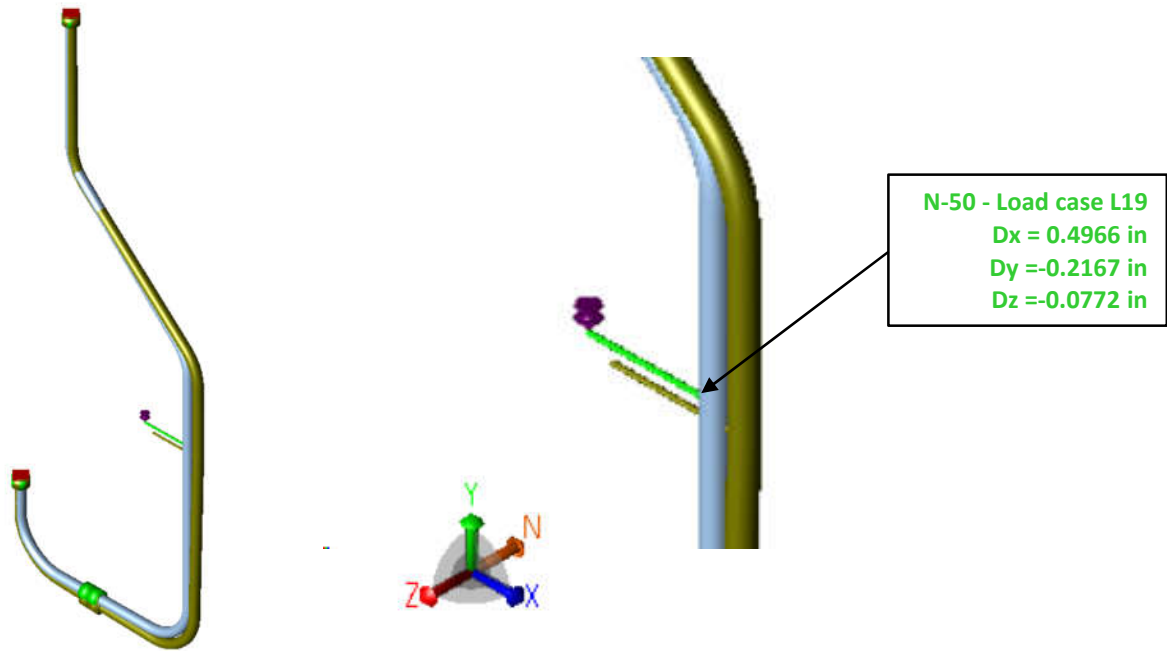


Fig-11. Node N-50 Maximum Displacements

Maximum displacement values were taken from all operating conditions design cases (L19). The values in the X, Y, and Z directions are shown above.

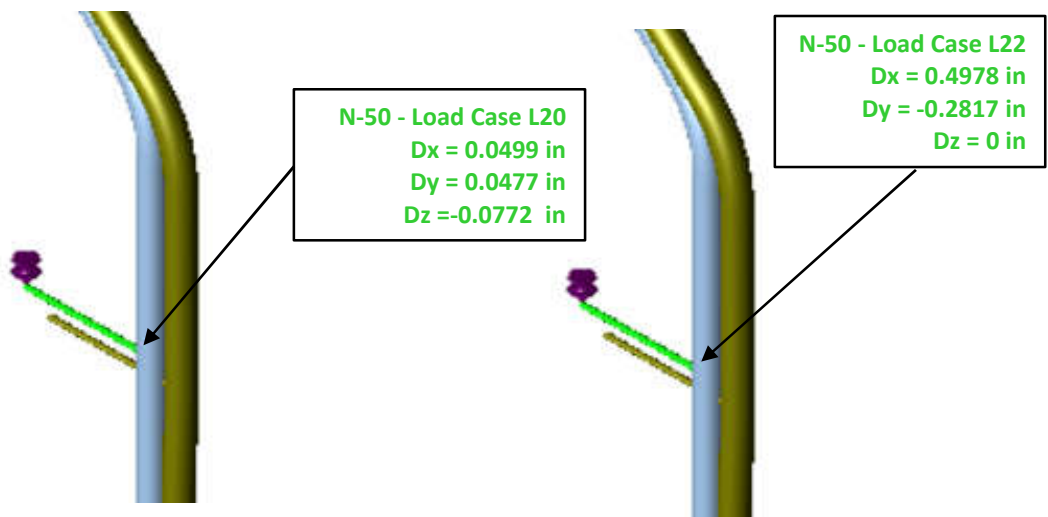


Fig-12. Node N-50 Maximum Displacements

Maximum displacement values were taken from all occasional cases (L20) and expansion case (L22). The values in the X, Y, and Z directions are shown above.

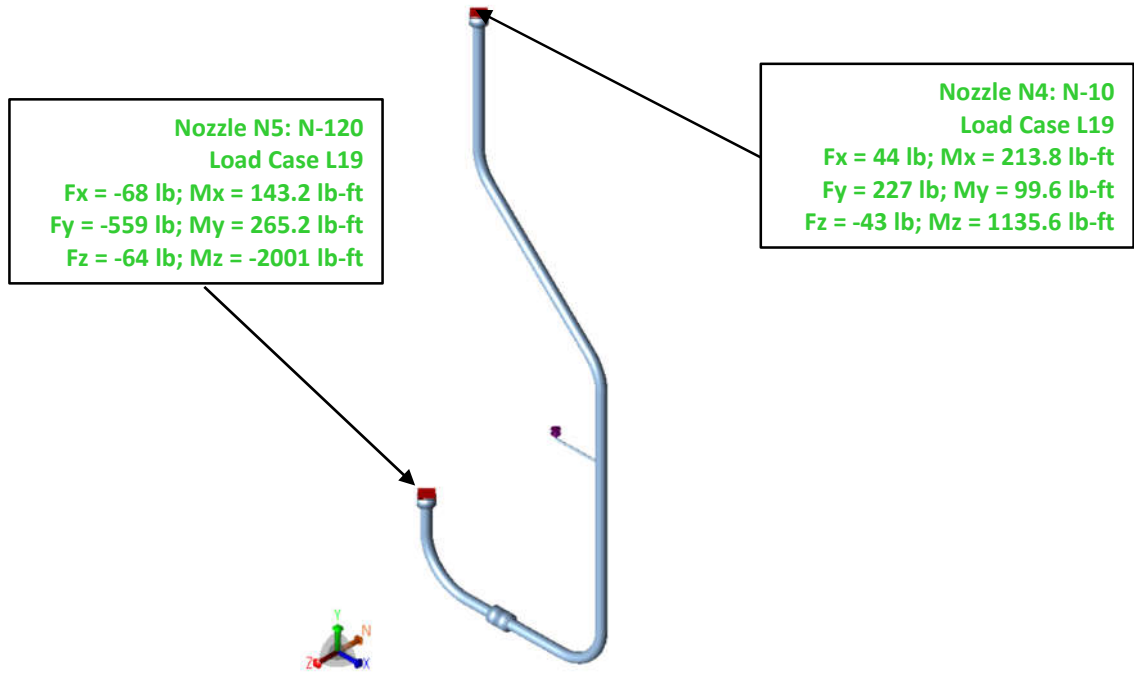


Fig-13. OPE Maximum Forces

Maximum nozzle load values were taken from all operating conditions design cases (L19). The values in the X, Y, and Z directions are shown above.

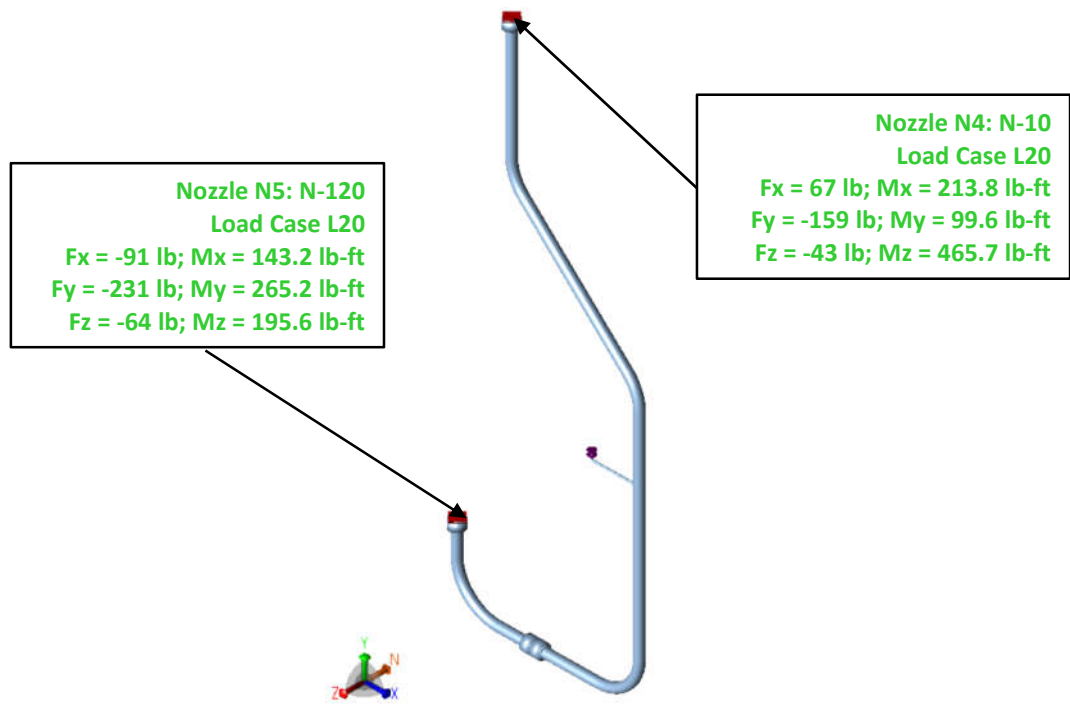


Fig-14. OCC Maximum Forces

Maximum nozzle load values were taken from all occasional cases (L20). The values in the X, Y, and Z directions are shown above.



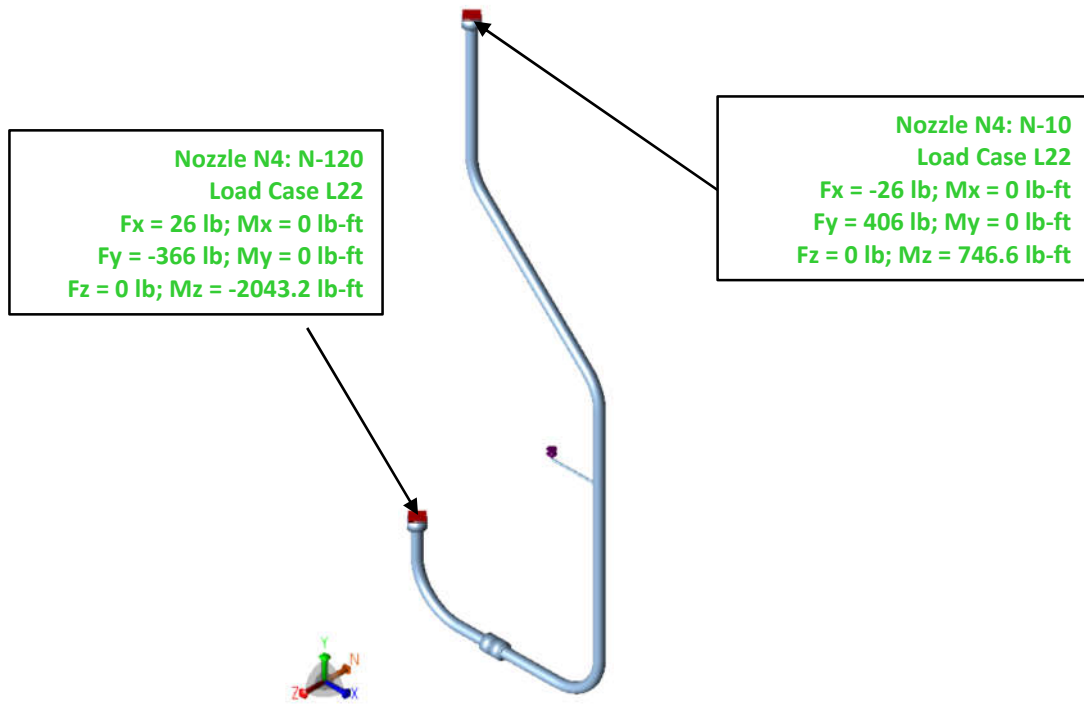


Fig-15. EXP Maximum Loads

Maximum nozzle load values were taken from expansion case ( L22). The values in the X, Y, and Z directions are shown above.

Node	Load Case	FX lb.	FY lb.	FZ lb.	MX ft.lb.	MY ft.lb.	MZ ft.lb.
N-10 / Nozzle N4		TYPE=Rigid ANC;					
	5(SUS)	25	-148	0	0	0	313.5
	19(OPE)	44	227	-43	213.8	99.6	1135.6
	20(OCC)	67	-159	-43	213.8	99.6	465.7
	21(EXP)	-23	364	0	0	0	670
	22(EXP)	-26	406	0	0	0	746.6
	<b>MAX</b>	<b>67/L20</b>	<b>406/L22</b>	<b>-43/L19</b>	<b>213.8/L19</b>	<b>99.6/L19</b>	<b>1135.6/L19</b>
N-120 / Nozzle N5		TYPE=Rigid ANC;					
	5(SUS)	-25	-221	0	0	0	14.1
	19(OPE)	-68	-559	-64	143.2	265.2	-2001
	20(OCC)	-91	-231	-64	143.2	265.2	195.6
	21(EXP)	23	-328	0	0	0	-1833.5
	22(EXP)	26	-366	0	0	0	-2043.2
	<b>MAX</b>	<b>-91/L20</b>	<b>-559/L19</b>	<b>-64/L19</b>	<b>143.2/L19</b>	<b>265.2/L19</b>	<b>-2043.2/L22</b>

Fig-26. Maximum Load at Nozzle N4 and N5

Maximum nozzle load values were taken from all 3 load cases (L18; L19 and L20). The values in the X, Y, and Z directions are shown above.

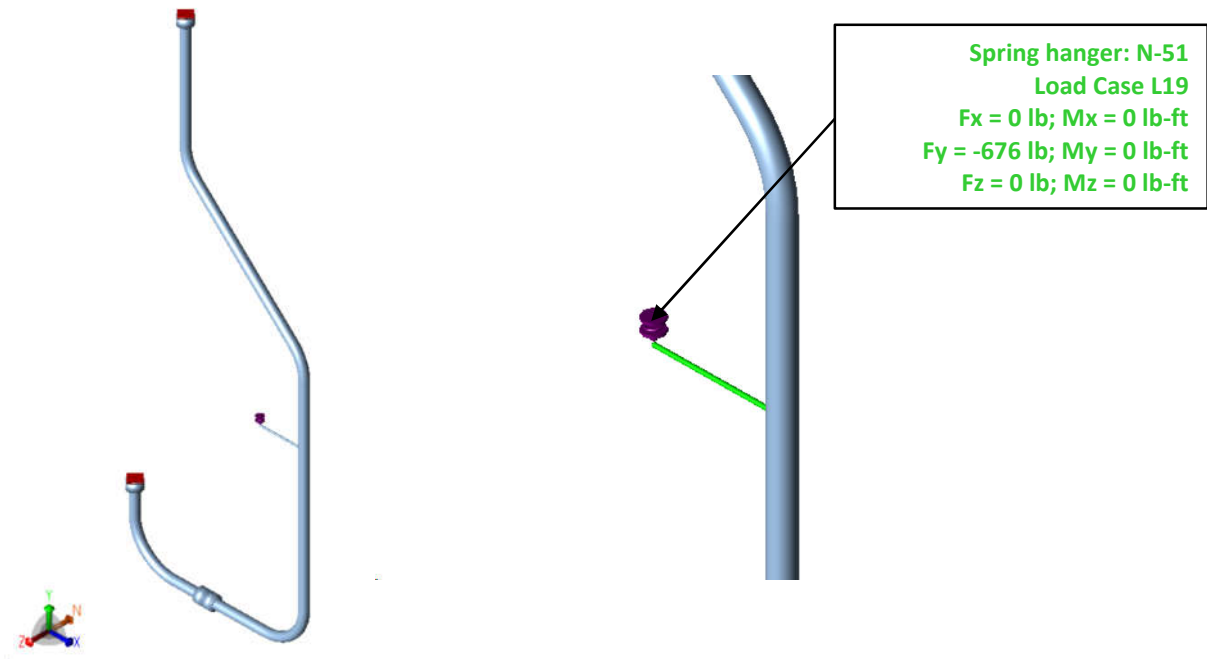


Fig-16. Spring Hanger Maximum Loads

Spring Hanger operation load values were taken from operation case ( L19). The values in the X, Y, and Z directions are shown above.

Node	Load Case	FX lb.	FY lb.	FZ lb.	MX ft.lb.	MY ft.lb.	MZ ft.lb.
N-51 / Spring Hanger		TYPE=Prog Design VSH;Tag=B-268, Size 8, Type F					
	5(SUS)	0	-639	0	0	0	0
	19(OPE)	0	-676	0	0	0	0
	20(OCC)	0	-641	0	0	0	0
	21(EXP)	0	-36	0	0	0	0
	22(EXP)	0	-40	0	0	0	0
	<b>MAX</b>	<b>0</b>	<b>-676/L19</b>	<b>0</b>	<b>0</b>	<b>0</b>	<b>0</b>

Spring Hanger Recommended Type, Size and loads are presented in the table below:

Hanger Table W/Text									
CAESAR II Ver.12.00.00.4000, (Build 200403) Date: MAR 2, 2021 Time: 11:8									
Job Name: 16579 - PIPING LINE N4-N5 WITH LOADS, SPRING HANGER AND TRUNNION									
Licensed To: Edit name in <system>\company.txt									
CAESAR II HANGER REPORT (TABLE DATA FROM DESIGN RUNS)									
NODE	NUM RQD	FIG. NO.	SIZE	VERTICAL MOVEMENT (in.)	HOT LOAD (lb.)	THEORETICAL INSTALLED LOAD (lb.)	ACTUAL INSTALLED LOAD (lb.)	SPRING RATE (lb./in.)	HORIZONTAL MOVEMENT (in.)
51	1	B-268	8	-0.125	675.	656.	0.	150.	0.425
ANVIL									
Tag=B-268;									
** VARIABLE SUPPORT SPRING DESIGNED ..... MID RANGE									
MINIMUM ALLOWED SINGLE SPRING LOAD ..... (lb.) 525.000									
MAXIMUM ALLOWED SINGLE SPRING LOAD ..... (lb.) 900.000									
RECOMMENDED INSTALLATION CLEARANCE ..... (in.) 12.125									

Fig-17. Spring Hanger Table - CEASAR II Output

## **LIMITS OF LIABILITY FOR PRESSURE VESSEL ENGINEERING**

This report was prepared through consultation with the client to obtain an accurate description of the working system and was designed using standard industry practices. All design data and the relevant loads were provided by the client.

The PSA models generated represent the drawings provided with the revision number identified in the first page and in the reference list of the report. All restraint recommendations must be followed for the system to be acceptable. Any change to the piping layout would alter the accuracy of the results. Pressure Vessel Engineering does not participate in the construction or fabrication and therefore cannot take responsibility for the as-built system.